FIRE SAFETY STUDY OF AIR DUCTS AND FIRE DAMPERS
EFFECT OF HANGAR SPACING, HANGAR SIZE, AND WALL THICKNESS
FIRST PROGRESS REPORT

Sponsored by:
American Iron and Steel Institute

U.S. DEPARTMENT OF COMMERCe
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3 Located at 5285 Port Royal Road, Springfield, Virginia 22151.
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Introduction

In order to develop information on the performance of air conditioning ducts constructed of sheet steel the present program was initiated under the sponsorship of the American Iron and Steel Institute. Phase I of this program is a study of the feasibility and the nature of the penetration of fire into a duct, either by breakthrough of the fire through the wall or by collapse of the duct.

The penetration of fire into a duct is probably a function of the size, shape, thickness of material, type of material and type of joints used in the duct construction. The possibility of collapse of a duct exposed to fire is believed to be a function of the hangar size spacing and type of joints used and the structural properties of the duct. For discussion we have used two terms: "collapse of the duct" which means an overall structural failure, including tearing apart of the steel or opening of the joints; and "buckling of the duct" which means an accordion type of collapse without rupture of the
steel or opening of the joints.

As the first phase of the program, on March 10, 1971, five air ducts constructed of galvanized steel sheet were subjected to a test following the standard time-temperature curve used in fire tests of building constructions (ASTM E119). This fire test was performed in the floor test furnace facility of the National Bureau of Standards in Washington, D.C. The object of this program was to study the effects of hangar size, hangar spacing, duct wall thickness and joint technique on the structural behavior of steel ducts exposed to a fire.

1.0 Construction

1.1 Duct Set-up

The test specimens were suspended on hangars and trapeze angles in accordance with the recommendations of SMACNA (1). At the time of test it was not possible to obtain, locally, No. 8 gauge wire, and so wire gauge No. 9 was used for ducts A, B and C. The difference in cross sectional area of No. 8 and 9 SWG is only about 15%. For ducts D and E, \( \frac{1}{2} \) inch diameter rods were used for the hangars. Quasi continuity to simulate a horizontal run of duct resting on several supports, was simulated by having the ducts overhang the supports and placing blocking at the ends of the duct between the top of the ducts and the furnace closure. This also served to restrain any upward deflections of the ends during the initial part of the test. See Figure 1 and 2 for erection and construction details.
1.2 Instrumentation

Each duct and hangar was instrumented with 16 thermocouples (B & S 24 gauge, 0.020 in diameter, chromel-alumel). Two groups of three thermocouples each had the thermocouples connected together in parallel, making a total of 12 thermocouple channels on each duct. See Figure 3 for the locations of the thermocouples.

Deflections were measured at the hangar and at the midspan of each duct. However, since the deflection points were located on the top of the ducts, these measurements were of limited value since localized buckling of the ducts occurred during the test. In future tests of this kind, the deflection points will be referenced differently.

The locations of the thermocouples are shown in Figure 3. The deflections at the duct hangars are shown in Figure 4, the uncorrected deflections at midspan of the ducts are shown in Figure 5, and the net deflections at midspan are shown in Figure 6.

2.0 Stresses in Ducts and Hangars

The stresses in the ducts and the hangars at the start of the test were low. The approximate stresses are summarized in Table 1.
### Table 1

<table>
<thead>
<tr>
<th>Duct</th>
<th>Max Bending Stress</th>
<th>Max Shear Stress</th>
<th>Hangar Stress</th>
<th>Wall Thickness</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>Support psi</td>
<td>Center psi</td>
<td>Support psi</td>
<td>psi</td>
</tr>
<tr>
<td>A</td>
<td>173</td>
<td>1938</td>
<td>47</td>
<td>3780</td>
</tr>
<tr>
<td>B</td>
<td>323</td>
<td>864</td>
<td>36</td>
<td>3360</td>
</tr>
<tr>
<td>C</td>
<td>173</td>
<td>370</td>
<td>24</td>
<td>2360</td>
</tr>
<tr>
<td>D</td>
<td>293</td>
<td>680</td>
<td>39</td>
<td>3340</td>
</tr>
<tr>
<td>E</td>
<td>323</td>
<td>864</td>
<td>36</td>
<td>2770</td>
</tr>
</tbody>
</table>

A stability analysis indicates that for the 24 gauge ducts the critical shear stress for buckling is of the order of 1400 psi, and for the 20 gauge ducts the critical shear stress is of the order of 3200 psi.

#### 3.0 Behavior of the Ducts During the Tests

3.1 Temperature Response

The average furnace temperatures are shown in Figure 7. The difficulty with the furnace temperature at the beginning of the test was due to a defective thermocouple circuit. However, this does not affect the results of this preliminary test. Generally, the temperature of the ducts and hangars followed the adjacent furnace temperature. See figures 8 to 12 for a comparison of the duct temperatures and the adjacent furnace temperatures. During this test the south end of the furnace was somewhat hotter than the north subsequently the furnace controls were adjusted to provide a more uniform temperature.
3.2 **Structural Response**

As the temperature of the ducts rose during the test, the sheet steel softened and the ducts sagged at midspan and over the trapeze supports. This sagging over the supports destroyed the simulated continuity and the ducts were simply supported for the rest of the test.

The ducts failed in the order of E, B, D, and A. Duct C did not fail during the test.

Duct E failed at 26 minutes by buckling collapse over the hangar supports.

Duct B failed at 28 minutes test time by generalized buckling of the sides of the duct.

Duct D, which was made of 20 gauge steel, failed at 51 minutes by failure of the joints. Because of the heavier gauge of this duct, the joints were made with pop rivets rather than slip joints. These pop rivets allowed the metal on each side of the joint to separate.

Duct A collapsed at 66 minutes.
Duct C showed only moderate bowing and sagging during the test. This duct had the shortest span between supports.

In general, the structural behavior improved with reduced span.

3.3 Joints
In every case, except duct C which did not fail, the ducts ultimately failed at the joints. Thus the joint construction is an important factor governing the performance of the duct.

4.0 Discussion and Recommendations
4.1 Discussion of Results
The behavior of the ducts indicated that with reduced hangar spacing and adequate joints a sheet steel metal duct can probably be designed to withstand a fire exposure of up to 2 hours without failure. Failure in this case meaning the inability of the duct to maintain its structural integrity and permitting fire to enter the duct.

4.2 Recommendation For Additional Research
The optimum spacing of the hangars, and the optimum design of the joints and ducts for these specified fire resistances can be determined only by additional tests. These additional tests should
include tests of joint strength and tests of other shapes, such as circular ducts, and insulated ducts. This test phase did not examine other significant factors such as the transfer of heat from a fire to air moving in the duct, the ability of a duct to transmit a fire from one compartment to another, and methods for preventing a fire from being transmitted from one compartment to another.
NORTH

2X4-20" LONG (~16" O.C.

WEST

JOISTS 2X8 @ 16" O.C.

18'

2 LAYERS 5/8" TYPE X GYPSUM BOARD
1 LAYER MONOKOTE ON METAL LATH

MARINITE BLOCKING

CAP

HANGAR

TC PIER

EAST

TYPICAL ERECTION DETAILS

DUCT PROGRAM
**Figure 2**

**DUCTS - ASSEMBLY DETAILS**

<table>
<thead>
<tr>
<th>TEST UNIT</th>
<th>SPAN &quot;L&quot;</th>
<th>FLAT OVERHANG-M</th>
<th>HANGAR*</th>
<th>GA</th>
</tr>
</thead>
<tbody>
<tr>
<td>A</td>
<td>10.0</td>
<td>1.0</td>
<td>9 GA</td>
<td>24</td>
</tr>
<tr>
<td>B</td>
<td>7.5</td>
<td>1.5</td>
<td>9 GA</td>
<td>24</td>
</tr>
<tr>
<td>C</td>
<td>5.0</td>
<td>1.0</td>
<td>9 GA</td>
<td>24</td>
</tr>
<tr>
<td>D</td>
<td>7.5</td>
<td>1.5</td>
<td>1/4</td>
<td>20</td>
</tr>
<tr>
<td>E</td>
<td>7.5</td>
<td>1.5</td>
<td>1/8</td>
<td>24</td>
</tr>
</tbody>
</table>

* TRAPEZE SHELF ANGLE - 1" x 1" x 1/8"

ALL DUCTS GALVANIZED
THREE THERMOCOUPLES MARKED NO. II AND THREE MARKED NO. 12 ARE CONNECTED IN PARALLEL.

DUCT TEST NO. 1
THERMOCOUPLE LOCATIONS
Figure 4

DEFLECTION vs. TIME

DEFLECTION AT DUCT HANGAR vs. TIME

A DUCT
B DUCT
C DUCT
D DUCT
E DUCT

WIRE BROKE

NO READINGS AFTER 80 MIN
Figure 5

DEFLECTION AT CENTER OF DUCTS vs. TIME

DEFLECTION — INCHES

TIME — MINUTES

NO READINGS AFTER 80 MIN

A DUCT
B DUCT
C DUCT
D DUCT
E DUCT
Figure 6

FIRST DUCT TEST
NET CENTERLINE DEFLECTIONS
VS.
TIME

NET DEFLECTION - INCHES

TEST TIME - MINUTES
Figure 7

AVERAGE FURNACE TEMPERATURE COMPARED WITH E119
Figure 8

COMPARISON OF DUCT MAX AND ADJACENT FURNACE TEMPERATURES TEST 485

DUCT TEMPERATURE

TEMPERATURE RISE (DEG C)

TIME (MINUTES)

ADJACENT FURNACE TEMPERATURE
Figure 9

Comparison of Duct and Adjacent Furnace Temperatures Test 486

Duct Temperature

Temperature Rise (Deg C)

Time (Minutes)

Adjacent Furnace Temperature
Figure 10

COMPARISON OF DUCT and ADJACENT FURNACE TEMPERATURES TEST 486

DUCT TEMPERATURE

ADJACENT FURNACE TEMPERATURE
Figure 11
COMPARISON OF DUCT TEMPERATURE AND ADJACENT FURNACE TEMPERATURES TEST 48C

DUCT TEMPERATURE

TEMPERATURE RISE (DEG C)

TIME (MINUTES)

ADJACENT FURNACE TEMPERATURE
Figure 12
Comparison of Duct Temperatures and Adjacent Furnace Temperatures Test 466

△ Duct Temperature

ADJACENT FURNACE TEMPERATURE

TIME (MINUTES)

TEMPERATURE RISE (DEG C)